

1 Problem

Equation 1 models the motion of a double pendulum for small oscillations about equilibrium with a perpendicular force acting at the end of the lower pendulum.

$$\begin{bmatrix} (m_1 + m_2)l_1^2 & m_2l_2l_1 \\ m_2l_1l_2 & m_2l_2^2 \end{bmatrix} \begin{Bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{Bmatrix} + \begin{bmatrix} (w_1 + w_2)l_1 & 0 \\ 0 & w_2l_2 \end{bmatrix} \begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = \begin{Bmatrix} fl_1 \\ fl_2 \end{Bmatrix} \quad (1)$$

For $m_1 = 1$ kg, $m_2 = 2$ kg, and $l_1 = l_2 = 0.5$ m perform the following engineering analysis tasks:

- State the characteristic equation.
- Calculate the eigenvalues and natural frequencies.
- Calculate the matrix of eigenvectors $[A]$.
- Normalize the eigenvectors such that $[A^*]^T[M][A^*] = [I]$ and confirm that your normalized eigenvectors satisfy $[A^*]^T[K][A^*] = [\Lambda]$, where $[\Lambda]$ is a diagonal matrix with the eigenvalues as diagonal elements.
- State the modal differential equations (including the modal force definitions) and the coordinate transformation from the modal coordinate model to physical coordinates.

2 Solution

- You start by substituting in the values given for m_1, m_2, l_1 , and l_2 (because solving for eigenvalues symbolically would be a real pain). The values for w_n is simply $m_n g$, in this case $g = 9.81$ m/s² because we are working in the metric system. When you perform this substitution we get the following:

$$[M]\{\ddot{\theta}\} + [K]\{\theta\} = f\{l\}$$

where:

$$[M] = \begin{bmatrix} 0.75 & 0.5 \\ 0.5 & 0.5 \end{bmatrix}$$

and:

$$[K] = \begin{bmatrix} 14.715 & 0 \\ 0 & 9.81 \end{bmatrix}$$

The characteristic equation is shown in Equation 2.

$$|-\lambda[M] + [K]| = 0 \quad (2)$$

Which, after some substitution is:

$$\begin{vmatrix} 14.715 - 0.75\lambda & -0.5\lambda \\ -0.5\lambda & 9.81 - 0.5\lambda \end{vmatrix} = 0$$

Using the definition of a determinant (which is denoted by the vertical bar, or pipe) we get:

$$0.125\lambda^2 - 14.715\lambda + 144.354 = 0$$

- Solving for λ yields: $\lambda_1 = 10.801$, and $\lambda_2 = 106.919$ (these should be ordered from lowest to highest), and since $\lambda \equiv \omega_n^2$, $\omega_{n_1} = 3.2865$, and $\omega_{n_2} = 10.3402$. Therefore the eigenvalues are:

$$\begin{bmatrix} 10.801 \\ 106.919 \end{bmatrix}$$

c. Apply Equation 3 for each value of λ while setting $a_1 = 1$ and find a_2 .

$$(-\lambda [M] + [K]) \{a\} = 0 \quad (3)$$

Which, of course is:

$$\begin{bmatrix} 14.715 - 0.75\lambda & -0.5\lambda \\ -0.5\lambda & 9.81 - 0.5\lambda \end{bmatrix} \begin{Bmatrix} a_1 \\ a_2 \end{Bmatrix} = 0$$

Solving this with λ_1 gives:

$$6.614 - 5.401a_2 = 0, \text{ or } a_2 = 1.225$$

Plugging in λ_2 gives:

$$-65.464 - 53.460a_2 = 0, \text{ or } a_2 = -1.225$$

Therefore the eigenvectors are:

$$[A] = \begin{bmatrix} 1 & 1 \\ 1.225 & -1.225 \end{bmatrix}$$

d. The next step is to find $[M_Q]$ which you do by applying Equation 4.

$$[M_Q] = [A]^T [M] [A] \quad (4)$$

Which evaluates to:

$$\begin{bmatrix} 1 & 1.225 \\ 1 & -1.225 \end{bmatrix} \begin{bmatrix} 0.75 & 0.5 \\ 0.5 & 0.5 \end{bmatrix} \begin{bmatrix} 1 & 1 \\ 1.225 & -1.225 \end{bmatrix} = [M_Q] = \begin{bmatrix} 2.725 & 0 \\ 0 & 0.275 \end{bmatrix}$$

Thus, $M_{Q_{11}} = 2.725$, and $M_{Q_{22}} = 0.275$. Equation 5 shows how to find the normalized eigenvectors:

$$[A^*] = \begin{bmatrix} \frac{a_{11}}{\sqrt{M_{Q_{11}}}} & \frac{a_{12}}{\sqrt{M_{Q_{22}}}} \\ \frac{a_{21}}{\sqrt{M_{Q_{11}}}} & \frac{a_{22}}{\sqrt{M_{Q_{22}}}} \end{bmatrix} \quad (5)$$

Therefore:

$$[A^*] = \begin{bmatrix} 0.606 & 1.906 \\ 0.742 & -2.334 \end{bmatrix}$$

To verify $[A^*]$ we use: $[A^*]^T [M] [A^*] = [I]$, and $[A^*]^T [K] [A^*] = [\Lambda]$

$$\begin{bmatrix} 0.606 & 0.742 \\ 1.906 & -2.334 \end{bmatrix} \begin{bmatrix} 0.75 & 0.5 \\ 0.5 & 0.5 \end{bmatrix} \begin{bmatrix} 0.606 & 1.906 \\ 0.742 & -2.334 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

and

$$\begin{bmatrix} 0.606 & 0.742 \\ 1.906 & -2.334 \end{bmatrix} \begin{bmatrix} 14.715 & 0 \\ 0 & 9.81 \end{bmatrix} \begin{bmatrix} 0.606 & 1.906 \\ 0.742 & -2.334 \end{bmatrix} = \begin{bmatrix} 10.801 & 0 \\ 0 & 106.919 \end{bmatrix}$$

e. In order to state the modal differential equations we must find the modal force definitions with Equation 6

$$[Q] = [A^*]^T f \{l\} \quad (6)$$

Which equals:

$$[Q] = \begin{bmatrix} 0.606 & 0.742 \\ 1.906 & -2.334 \end{bmatrix} \begin{Bmatrix} 0.5f \\ 0.5f \end{Bmatrix} = \begin{Bmatrix} 0.674f \\ -0.214f \end{Bmatrix}$$

Thus the modal differential equations are:

$$\ddot{q}_1 + 10.801q_1 = 0.674f, \text{ and } \ddot{q}_2 + 106.919q_2 = -0.214f$$

And to get back to the physical coordinates use Equation 7 after solving for q_1 , and q_2 .

$$\begin{Bmatrix} \theta_1 \\ \theta_2 \end{Bmatrix} = [A^*] \begin{Bmatrix} q_1 \\ q_2 \end{Bmatrix} \quad (7)$$